

**GUJARAT TECHNOLOGICAL UNIVERSITY**  
**BE - SEMESTER-VI • EXAMINATION – SUMMER • 2014**

**Subject Code: 160201****Date: 19-05-2014****Subject Name: Automobile Component Design****Time: 10:30 am - 01:00 pm****Total Marks: 70****Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.

**Q.1 (a)** What is herringbone helical gear? What is the difference between double helical gear and herringbone helical gear? **04**

**(b)** A pair of straight teeth spur gears, having  $20^{\circ}$  involute full depth teeth is to transmit 12.5 kW at 300 r.p.m. of the pinion. The speed ratio is 3:1. The allowable static stresses for gear of cast iron and pinion of steel are 65 MPa and 110 MPa respectively. Assume the following:

No. of teeth of pinion = 18, Face width = 14 times module,

velocity factor  $C_v = \frac{4.5}{4.5+V}$ ,  $V$  = pitch line velocity in m/s,

and tooth form factor  $(y) = 0.154 - \frac{0.912}{\text{No. of teeth}}$

$\sigma_{es} = 600$  MPa,  $E_p = 200$  kN/mm<sup>2</sup> and  $E_g = 200$  kN/mm<sup>2</sup>.

Wear load =  $d_p \cdot f \cdot KW \cdot Q$

Determine the module, face width and pitch circle diameter of gears. Check the gears for wear.

**Q.2 (a)** A ball bearing operates on the following work cycle. **07**

Element No.	Radial load (N)	Speed (rpm)	Element time (%)
1	3000	720	30
2	7000	1440	50
3	5000	900	20

The dynamic load capacity of the bearing is 16.6 kN. Calculate

- i. The average speed of rotation,
  - ii. The equivalent radial load and
  - iii. The bearing life in hrs.
- (b)** State and explain design consideration of casting with figure. **07**

**OR**

**(b)** Explain the procedure of designing multi speed automobile gear box. **07**

**Q.3 (a)** What is the objective of preloading of rolling contact bearing? Where do you use preloaded rolling contact bearings? **04**

- (b) A pair of bevel gears connects two shafts at right angles and transmits 9 kW. Determine the required module and gear diameters for the following specifications. Check the gears for wear load. **10**

Particulars	Pinion	Gear
No. of teeth	22	60
Material	Semi steel	Grey cast iron
Allowable static stress	85 MPa	55 MPa
Speed (rpm)	1145	420
Tooth profile	$14\frac{1}{2}^{\circ}$ composite	$14\frac{1}{2}^{\circ}$ composite
BHN	200	160

**OR**

- Q.3 (a)** What is creep? How will you design the parts subjected to creep? Give the practical applications of the components designed for creep. **07**
- (b)** A pair of worm and worm wheel is designated as 3/60/10/6. The worm is transmitting 5.5 kW power at 1440 rpm to the worm wheel. The coefficient of friction is 0.1 and the normal pressure angle is  $20^{\circ}$ . Determine the components of the gear tooth force acting on the worm and the worm wheel. **07**
- Q.4 (a)** What are the advantages and disadvantages of “Wet Liner” and “Dry Liner” in I.C. engines? What are the desirable properties of cylinder materials? **07**
- (b)** The cylinder of a four-stroke diesel engine has the following specification. **07**  
 Brake power = 7 kW, Speed = 1000 rpm, Indicated mean effective pressure = 0.35 MPa, Mechanical efficiency = 80%,  
 Max. gas pressure = 3.5 MPa. The cylinder head is made of grey cast iron FG260 ( $S_{ut} = 260 \text{ N/mm}^2$  and  $\mu = 0.25$ ). The studs are made of plain-carbon steel 40C8 ( $S_{yt} = 380 \text{ N/mm}^2$ ). The factor of safety for all parts is 6. Calculate
- Bore and length of the cylinder liner.
  - Thickness of the cylinder head
  - Size, number and pitch of studs.

**OR**

- Q.4 (a)** Explain the piston materials. **04**

- (b) Design a cast iron piston for a single acting four stroke engine for the following data: **10**
- Cylinder bore = 300 mm; Stroke = 1.5D; Speed = 300r.p.m.  
 Indicated mean effective pressure = 0.85 N/mm<sup>2</sup>,  
 Maximum gas pressure = 5 N/mm<sup>2</sup>  
 Fuel consumption = 0.30 kg per BP per hour;  
 Mechanical efficiency = 80%;  
 Higher calorific value of fuel = 46000 KJ/kg;  
 Taking Permissible bending or tensile stress for C.I = 40 N/mm<sup>2</sup>;  
 Heat conductivity factor  $k$  for C.I. = 46.6 W/m/°C, and Temperature difference at the centre of the piston head and temperature at the edges piston head is = 220°C;  $C$  = Constant representing that portion of the heat supplied to the engine which is absorbed by the piston is 0.05;  
 The allowable radial Pressure of the gas on cylinder wall is 0.035;  
 The allowable tensile or bending stress for C.I ring is 90 N/mm<sup>2</sup> ;  
 The number of compression rings is 3 and there is one oil ring.  
 The allowable bearing pressure on skirt portion of the piston is 0.45 MPa;  
 The ratio of side thrust on liner to maximum gas load on piston ( $\mu$ ) is = 0.1;  
 Bearing pressure at small end of connecting rod is 30 MPa;  
 Bending stress for piston pin is 140 N/mm<sup>2</sup>; The length of the piston pin in the bush of the small end of the connecting rod is (0.45 D).

- Q.5 (a)** Why do inlet and exhaust valves have conical heads and seats? **04**  
**(b)** Explain with neat sketch, the design procedure for the connecting rod for an I.C. engine. **10**

**OR**

- Q.5 (a)** What is the difference between centre and overhung crankshafts? Where do you use them? Name the materials for crankshaft. **07**  
**(b)** Explain different modes of gear teeth failures, stating their reasons and remedies. **07**

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